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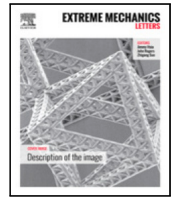
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Experiment-informed finite-strain inverse design of spinodal metamaterials

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ABSTRACT

Spinodal metamaterials, with architectures inspired by natural phase-separation processes, have presented a significant alternative to periodic and symmetric morphologies when designing mechanical metamaterials with extreme performance. While their elastic mechanical properties have been systematically determined, their large-deformation, nonlinear responses have been challenging to predict and design, in part due to limited data sets and the need for complex nonlinear simulations. This work presents a novel physics-enhanced machine learning (ML) and optimization framework tailored to address the challenges of designing intricate spinodal metamaterials with customized mechanical properties in large-deformation scenarios where computational modeling is restrictive and experimental data is sparse. By utilizing large-deformation experimental data directly, this approach facilitates the inverse design of spinodal structures with precise finite-strain mechanical responses. The framework sheds light on instability-induced pattern formation in spinodal metamaterials—observed experimentally and in selected nonlinear simulations—leveraging physics-based inductive biases in the form of nonconvex energetic potentials. Altogether, this combined ML, experimental, and computational effort provides a route for efficient and accurate design of complex spinodal metamaterials for large-deformation scenarios where energy absorption and prediction of nonlinear failure mechanisms is essential.

1. Introduction

The rapid advancement of resolution and throughput in additive manufacturing has opened doors to new possibilities in engineering mechanical metamaterials (or architected materials) with properties that were once unattainable using conventional manufacturing techniques [1–5]. These properties include, e.g., high strength-to-weight ratios [2,6], negative Poisson's ratios [7], mechanical cloaking abilities [8], tailorable anisotropic stiffness [9–12], and high energy absorption [13,14]. While explorations on mechanical metamaterials have primarily focused on periodic and symmetric truss- [15] and plate-based [16] lattices, recently more attention has been given to shell-based morphologies. These morphologies, such as triply periodic minimal surfaces [17,18] and spinodal architectures [9,19–21] have gained traction in the field as they do not possess nodes or joints, behave in a mechanically efficient manner due to their doubly curved shells, and inherently mitigate stress concentrations.

Spinodal-like—or *spinodoid*—metamaterials are especially intriguing because they possess an aperiodic and asymmetric microstructure resembling the morphologies observed during the early stages of spinodal decomposition or rapid diffusion-driven phase separation in a homogeneous mixture of immiscible phases (Fig. 1). These nature-inspired designs spanning multiple length scales—from nanoscale to macroscale—can either be manufactured using scalable self-assembly via polymerization-induced phase separation in polymer blends [19] or additive manufacturing of morphologies extracted from phase separation simulations [20,21]. In contrast to truss-, plate- and shell-based lattices, the resulting smooth and bicontinuous topologies enable robustness to manufacturing defects, mitigate any harmful stress concentrations, and exhibit extreme mechanical resilience [19,21]. *In silico* tuning of the underlying energetics of the spinodal decomposition process opens up a diverse design space of topologies and corresponding mechanical properties [9,20,22], leading to a recent flurry of proposed spinodal metamaterials for ultralight structures [19,23–25], energy absorption [21,26,27], bone-mimetic implants [9,28,29], acoustics [30], mass transport [31], among other applications.

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Despite the recent advances, the structure–property relations of spinodal metamaterials have primarily been explored only in a forward fashion—computational or experimental mechanical analyses of a few representative samples chosen through trial-and-error or intuition of the design space [19,21,23,24,26,27,32]. Furthermore, clear links between specific features of the morphology and the ensuing mechanical response are lacking. In light of the vast design space of spinodal metamaterials, the question of interest is that of inverse design, i.e., how to efficiently identify designs with tailored properties dictated by stress–strain responses. While machine learning (ML) has recently seen some success in inverse design of metamaterials [9–12,22, 24,33–39]—including but not limited to spinodal metamaterials—these algorithms face a challenge of quality–quantity duality.

High-throughput simulations of mechanical responses yield large quantities of data, which are of sufficient fidelity only in linear and small-strain regimes. For instance, in the context of spinodal metamaterials, ML models have been trained on *in silico* data to inversely design for linear stiffness tensor components and tailored anisotropy [9]. However, in the case of finite-strain behaviors, responses due to phenomena such as material and geometric nonlinearities, buckling, self-contact, dissipation, fracture, and meshing artifacts add significant complexity and computational cost. On the contrary, while experimental data can serve as the ground-truth, highest-fidelity data for ML models, the quantity of data is severely limited by the number of time-consuming and costly experiments that can be reasonably performed.

Here, we propose a direct experiment-to-ML inverse design framework to design spinodal metamaterials with tailored finite-strain mechanical responses. We create a reasonably sized dataset of 107 shell-based spinodal metamaterial architectures along with their stress–strain responses to 40% strain along three principal directions—efficiently obtained experimentally via *ex situ* and *in situ* uniaxial compression at the microscale. The ML framework consists of a forward model that surrogates the structure–property relations and an inverse optimization scheme that finds the designs for a target large-deformation stress–strain response. To address the quality–quantity duality challenge, we introduce a new ML architecture that uses physics-enhanced inductive biases to eliminate the need for large-quantity and low-quality simulation data. To complement these predictions, we employ nonlinear finite element models that capture architecture-dependent deformation mechanisms, and shed light on the sources of energy absorption upon large deformation. In agreement with experiments and predictions, these simulations explain the relation between the directional curvature distribution and predicted energy absorption metrics. Furthermore, we demonstrate the ML framework's ability to identify a spinodal morphology that exhibits a target behavior that lies outside the training data domain. Altogether, this experiment-informed ML effort closes the gap between the structure–property relations of spinodal metamaterials in a large-deformation regime, particularly by accounting for nonlinear responses due to deformation mechanisms in complex architectures.

2. Results and discussion

2.1. Spinodal morphology design space

To replicate a variety of morphologies obtained through spinodal decomposition, such as those obtained in diffusion-driven phase separation of a homogeneous mixture of immiscible phases, our first step is to define a design space that parametrizes all possible morphologies. Cahn [40] demonstrated that the resulting phase field solution to the canonical Cahn–Hilliard equation can be approximated by a Gaussian random field (GRF), i.e., a superposition of a large number of standing waves with a narrow band of similar wavenumbers. In Fourier space, this corresponds to a spectral density function (SDF) given by a diffused spherical surface of radius equal to the wavenumber (denoted by β henceforth) and centered at the origin. Inspired from the formalization

of this approximation [31,41], we construct the phase field $\varphi : \Omega \to \mathbb{R}$ in a domain $\Omega \subset \mathbb{R}^3$ directly in Fourier space as

$$\varphi = \mathcal{F}^{-1}\left[\mathcal{F}[\varphi_{\text{noise}}] \odot \mathcal{F}[\varphi_{\text{filter}}]\right] = \mathcal{F}^{-1}\left[\mathcal{F}[\varphi_{\text{noise}}] \odot \left(\rho[\varphi_{\text{filter}}]\right)^{1/2}\right]. \quad (1)$$

Here, $\mathcal{F}[\cdot]$, $\mathcal{F}[\cdot]^{-1}$, and $\rho[\cdot]$ denote the Fourier transform, inverse Fourier transform, and SDF (squared magnitude of the Fourier transform), respectively, while $\odot$ denotes the Hadamard product, φ_{noise} represents the initial phase field of a homogeneous mixture of immiscible phases as an independent and identically distributed standard Gaussian noise (i.e., zero mean and unit variance). Convolution of φ_{noise} with φ_{filter} (equivalently, Hadamard product in the Fourier space) represents the phase separation process that transforms the homogeneous mixture φ_{noise} into the phase-separated phase field φ. We define φ_{filter} through its SDF as

$$\rho[\varphi_{\text{filter}}](\boldsymbol{k}) = \underbrace{\exp\left(-\frac{(r-\beta)^2}{2\lambda_r^2}\right)}_{\text{wavenumber control}} \odot \underbrace{\sum_{i=1}^{3} \sigma\left(-\lambda_\phi(\phi_i - \theta_i)\right)}_{\text{anisotropy control}}$$

$$\text{with} \quad \phi_i = \min\{\cos^{-1}(k_i/r), -\cos^{-1}(k_i/r)\} \quad \text{and} \quad r = \|\boldsymbol{k}\|, \quad (2)$$

where $\boldsymbol{k}$ denotes a wave-vector in the Fourier space; with $\lambda_r > 0$ and $\lambda_\phi > 0$ as constants. The first term—labeled *wavenumber control*—specifies that the spectral density is limited to a narrow Gaussian band with mean wavenumber β and standard deviation λ_r. Choosing a higher β yields morphologies with a finer microstructural lengthscale. The second term—labeled *anisotropy control*—adds directional constraints to the wave-vectors. Specifically, the wave-vectors' orientations (given by ϕ_i) are constrained to cones centered at the origin and along the principal axes $\{\hat{e}_1, \hat{e}_2, \hat{e}_3\}$ with half-angles $\{\theta_1, \theta_2, \theta_3\}$, respectively (Fig. 1, center). However, instead of a hard limit, we relax the constraint by using a smooth sigmoid-type function, which in this case is $\sigma(\cdot) = (1 + \tanh(\cdot))/2$. Consequently, the probability of a wave-vector outside the cones decreases fast but smoothly with the rate determined by λ_ϕ. We add both radial and angular smoothing in the SDF to mitigate formation of non-smooth artifacts in the generated morphology (see Supplementary Fig. S1). We highlight that this model of anisotropy control is not just a mathematical construct; rather it serves as an approximation to the canonical Cahn–Hilliard equation with anisotropic mobility [9].

Choosing a cubic domain Ω of size $\ell \times \ell \times \ell$, the Fourier transforms in Eq. (1) are performed on a uniform grid of Ω with $X \times X \times X$ resolution, automatically ensuring triple periodicity in the generated structure. The morphology of the spinodal metamaterial is obtained by computing the zero level-set of φ, i.e., for $\boldsymbol{x} \in \Omega$, followed by extruding the surface pointwise along both inward and outward normal directions equally for a final surface thickness of $h \ll \ell$. Note that due to the randomness of φ_{noise}, the resulting structures are stochastic and hence two morphology realizations for the same design parameters may be different. To reduce the effects of stochasticity, we choose $\beta\ell$ and h/ℓ to be sufficiently high and low, respectively, to ensure separation of scales between the microstructural length scale and the domain dimensions. We validate the choice in β and h through a systematic computational homogenization analysis as well as manufacturing considerations, respectively (Supplementary Note 2).

For the scope of this work, we uniquely defined each spinodal metamaterial design by the design parameters $\boldsymbol{\Theta} = [\theta_1, \theta_2, \theta_3]$ with $\theta_1, \theta_2, \theta_3 \in \{0\} \cup [\theta_{\min}, \theta_{\max}]$ (i.e., the cone angles), while keeping the remaining parameters constant (see Supplementary Table S1). The angles, when non-zero, are lower-bounded by $\theta_{\min} = 20°$ to ensure bicontinuity of the structures [9] and upper-bounded by $\theta_{\max} = 70°$ to avoid degenerate (almost) isotropic structures. For instance, when $\boldsymbol{\Theta} = [0, 0, \theta_3]$, this one-dimensional (1D) parameter subspace results in structures with distinctive lamellar-like features (Fig. 1, left). In contrast,
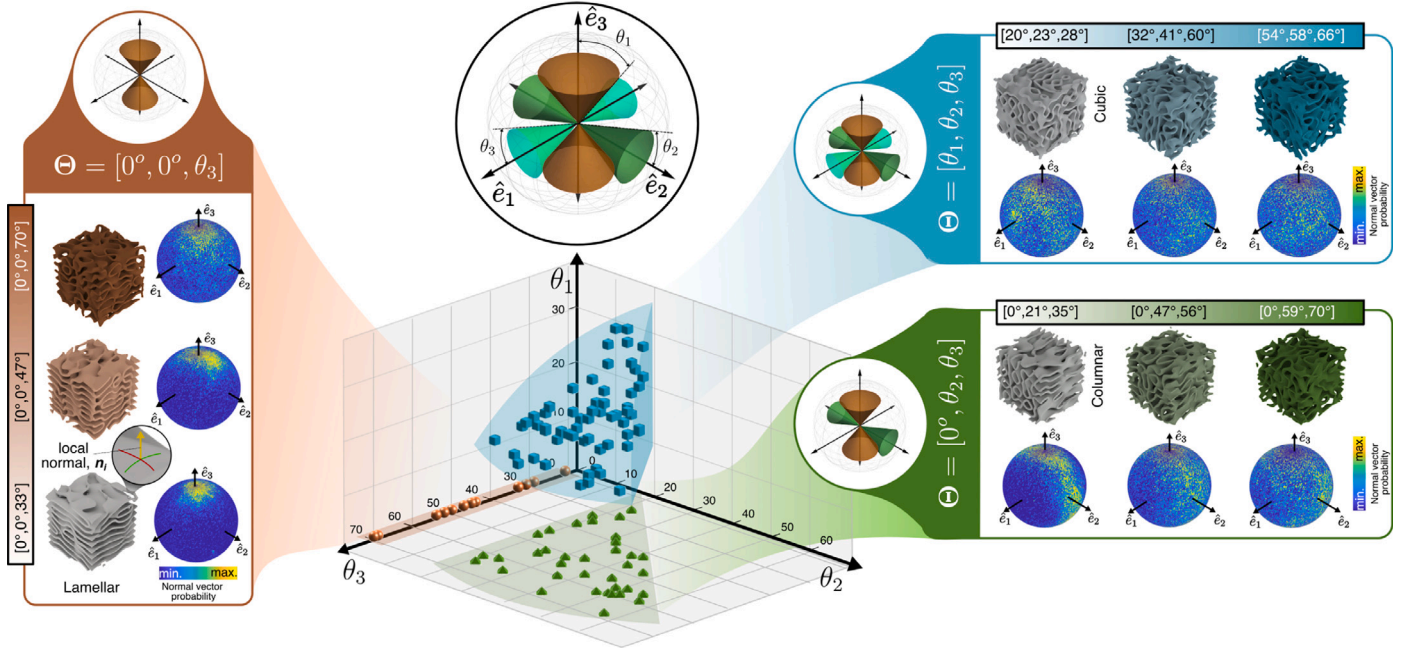
Fig. 1. Spinodal morphology design space defined by a three-parameter Θ representation (center). Three design subspaces, defined by the non-zero dimensionality of the $\Theta = [\theta_1, \theta_2, \theta_3]$ vector are represented by color clouds encompassing the designs used for training. The one-dimensional (1D) subspace corresponding to *lamellar* morphologies was represented by a non-zero θ_3, while the 2D and 3D subspaces subsequently added non-zero θ_2 and θ_3 parameters and corresponded to *columnar* and *cubic* morphologies, respectively. Three representative designs of increasing norm $|\Theta|$ are presented within each subspace, along with pole figures denoting the directional probability of normal vectors $\boldsymbol{n}$.

the corresponding 2D and 3D parameter subspaces—$\Theta = [0, \theta_2, \theta_3]$ and $\Theta = [\theta_1, \theta_2, \theta_3]$—exhibit columnar- or cubic-like features, respectively (Fig. 1, right). The resulting morphologies can be characterized by their corresponding surface-normal distributions, represented as spherical pole figures in Fig. 1, indicating the directional distribution of material curvature within the morphologies. Overall, the design space admits a large and diverse set of morphological anisotropy, with the aim of linking said structures to their unique mechanical responses.

Notably, increases in the magnitudes of the θ_i parameters, approaching the theoretical maximum of 90°, results in isotropically distributed wave-vectors and the anisotropic structural distinctions disappear. However, for the sake of clarity in the subsequent discussions, we use the terminology of *lamellar*, *columnar*, and *cubic* structures when referring to the 1D, 2D, and 3D subspaces of Θ, respectively, regardless of their absolute values.

2.2. Dataset generation via nanomechanical experiments

Sampling from the design space defined above, we generated a dataset consisting of $N = 107$ spinodal morphologies by randomly sampling Θ. The dataset included 11 lamellar ($\Theta = [0, 0, \theta_3]$), 36 columnar ($\Theta = [0, \theta_2, \theta_3]$), and 60 cubic ($\Theta = [\theta_1, \theta_2, \theta_3]$, with $\theta_1 < \theta_2 < \theta_3$) morphologies. To augment our experimental effort, we removed redundancy in data generation due to permutation symmetry in the parameterization (i.e., the response along $\hat{e}_1$ for $[\theta_1, \theta_2, \theta_3]$ is equivalent to the response along $\hat{e}_2$ for $[\theta_2, \theta_1, \theta_3]$). To assess the response along the three principal directions for each of the unique N morphologies, we fabricated 321 samples corresponding to 107 geometries, printed in three orientations. Altogether, these samples represented 609 effective Θ parameterizations within our design space, accounting for permutation symmetry (see Supplementary Note 1). The 321 samples were fabricated out of IP-Dip photoresist using a two-photon lithography process (Supplementary Note 2), resulting in cubic unit cells with an average edge length of $\ell = 92 \pm 1.2$ μm and shell thickness of $h = 2.6 \pm 0.1$ μm, with an approximate relative density (i.e., fill fraction) of 40%. Using sub-micron resolution X-ray computed tomography, we

validated the geometric validity of our fabricated samples, ensuring accurate representation of curvatures, shell thicknesses, and relative densities (Supplementary Note 2).

We note that, imperfections due to additive manufacturing can affect the mechanical properties [42–44]. However, we did not aim to train the ML surrogate model on data whose imperfections have been artificially reduced through repeated experiments. Instead, we let the model account for the role of as-fabricated imperfections in the mechanical response.

To obtain the finite-strain response of each sample in the dataset, we performed both *ex situ* and *in situ* quasi-static uniaxial compression experiments (strain rate of 10^{-3} s^{-1}) along the three principal directions $\hat{e}_1$, $\hat{e}_2$, and $\hat{e}_3$. *In situ* observation of the compression enabled visualization of multiple nonlinear and irreversible mechanisms such as plastic buckling, self-contact, and fracture through the thickness of the shells (Fig. 2a)—which could be linked to specific characteristics of the large-deformation stress–strain response. The loading portion of the measured 321 stress–strain curves was used to train the ML model for inverse design, while the unloading portion was omitted since it carried minimal information in this large-strain regime (full loading–unloading responses are presented in Supplementary Note 1). To facilitate comparison to spinodal metamaterials made of similar polymeric constituents, we report the stress normalized by the elastic modulus E_s (3.2 ± 0.3 GPa) of IP-Dip photoresist, determined from experiments on monolithic micropillars with the same print parameters as the metamaterials.

As an appropriate qualitative indicator of mechanical anisotropy within each design, we represented the surface normal distributions in the form of a spherical pole figure, indicating the relative orientation of curved shells with respect to a loading direction (Fig. 2b). Regions with a higher distribution of surface normals correlated to a more compliant response in the linear regime, followed by an on-average lower stress level throughout subsequent deformation. In the cases of lamellar and columnar morphologies, directions with low surface-normal distributions tended to exhibit plastic buckling beyond the onset of nonlinearity (at strains of $\varepsilon \approx 5\%$) and consequently, negative-stiffness responses up
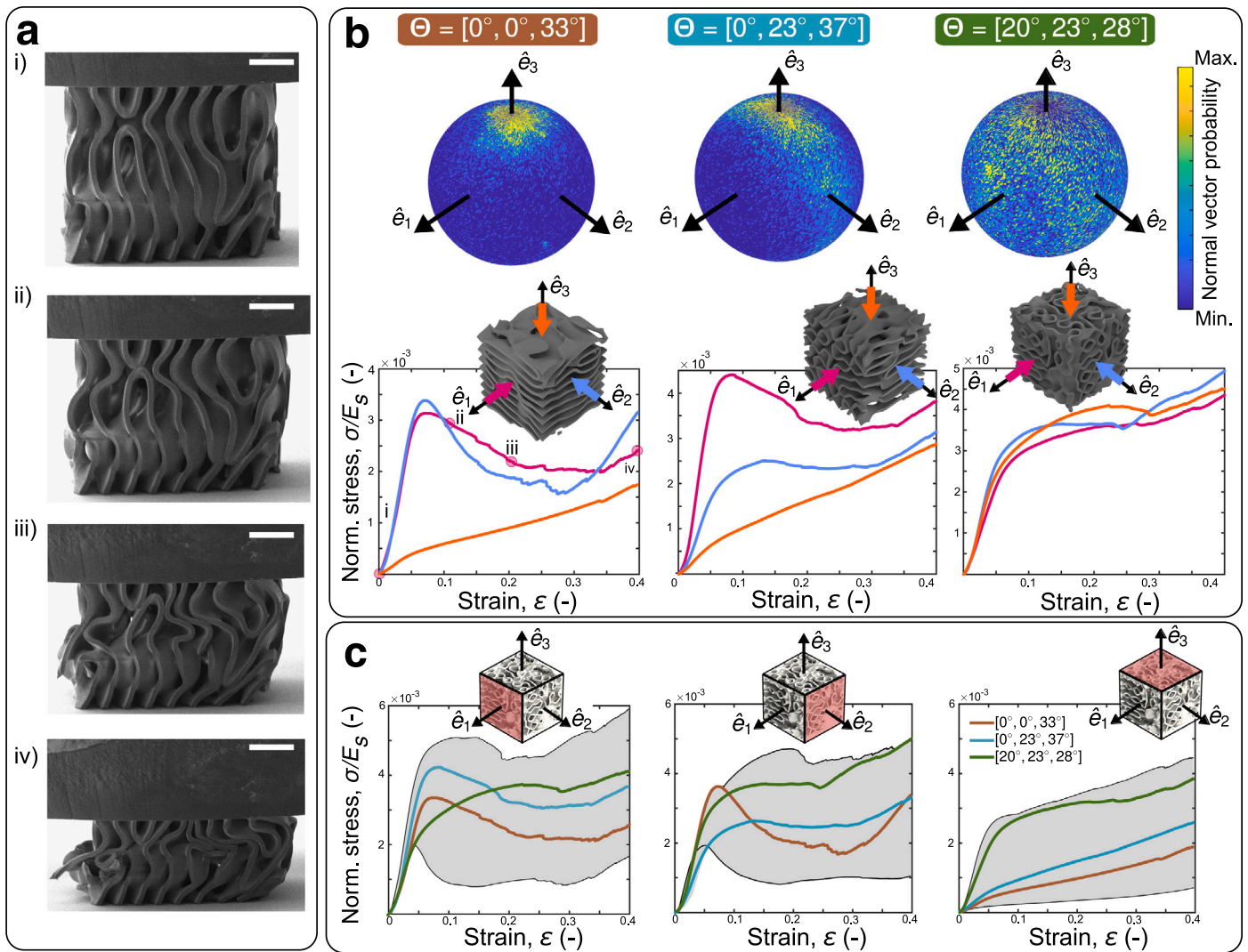
Fig. 2. Results and analysis of nanomechanical experiments. (**a**) *In situ* snapshots *i–iv* showing the progression of deformation for the $\Theta = [0°, 0°, 33°]$ morphology along the $\hat{e}_1$ direction, up to 40% strain. Scale bars, 20 μm. (**b**) Qualitative structure-to-response relations enabled by spherical pole figures denoting the directional surface-normal distributions for representative *lamellar* (left), *columnar* (center), and *cubic* (right) morphologies—accompanied by corresponding stress–strain responses along the $\hat{e}_i$ directions. The pole figures serve as a proxy for structural anisotropy, with higher surface-normal distributions along a given direction correlating to a more compliant response. The lamellar and columnar morphologies exhibited negative-stiffness regions corresponding to nonlinear buckling (as marked for the lamellar sample shown in (a)), along with stiffening at large deformations due to self-contact of shells. (**c**) Range of finite stress–strain behaviors across the training data as observed from *ex situ* compressions along the three principal directions $\hat{e}_1$ (left), $\hat{e}_2$ (center), $\hat{e}_3$ (right) highlighted on a generic spinodal morphology. The black lines denote the stress bounds across the training dataset, while color-coded responses correspond to the three representative morphologies shown in (b).

to strains of $\varepsilon \approx 20\%$ as observed via *in situ* experiments on the $\Theta = [0°, 0°, 33°]$ morphology (Fig. 2a,b left). Notably, the cubic morphologies approach similar responses when loading in all three directions with $\theta_1 \approx \theta_2 \approx \theta_3$. Further *in situ* observations for representative samples presented in Fig. 2b are included in Supplementary Note 2 (see Supporting Information Movies S1–S9).

When analyzing the dataset as a whole—simply represented in Fig. 2c as bounded by maximum and minimum responses with some highlighted morphologies within—we identify the buckling behavior to be prevalent in other lamellar morphologies as well as some low-angle columnar morphologies (i.e., when $\theta_2 < 30°$). For the subset of fabricated samples, we identified a distinctively different response in $\hat{e}_3$-loaded responses, primarily exhibiting monotonically increasing stress levels—a consequence of our $\theta_1 < \theta_2 < \theta_3$ criterion when selecting morphologies. We highlight that between the strains of $\varepsilon \approx 20\%$ to 40%, micro-cracks began to form causing fracture events that manifested as fluctuations in the stress response. Defining the energy absorbed as the integral of the stress–strain responses to 40% strain provided a broad distribution of performance metrics as a function of

morphology and orientation. Altogether, this comprehensive dataset sheds light on the complex nonlinear responses of high-relative-density spinodal morphologies, identifying a correlation between mechanisms such as buckling and self contact to qualitative morphology classifications. These observations add intuition to previously observed nonlinear responses in spinodal morphologies, while alternate mechanics-driven computational tools are required to identify structure–property relations in this highly nonlinear regime.

2.3. Forward modeling via physics-enhanced deep learning

Learning the highly nonlinear map from the design parameters Θ to the direction-dependent stress–strain responses of spinodal metamaterials would require a significant amount of data. To circumvent this issue and to work with limited experimental data available, we introduce a physics-enhanced ML framework that serves as a surrogate to the forward structure-to-property relations.

In the exemplar stress–strain behaviors in Fig. 2b, we observe nonlinear features that are typical of instabilities and pattern formation
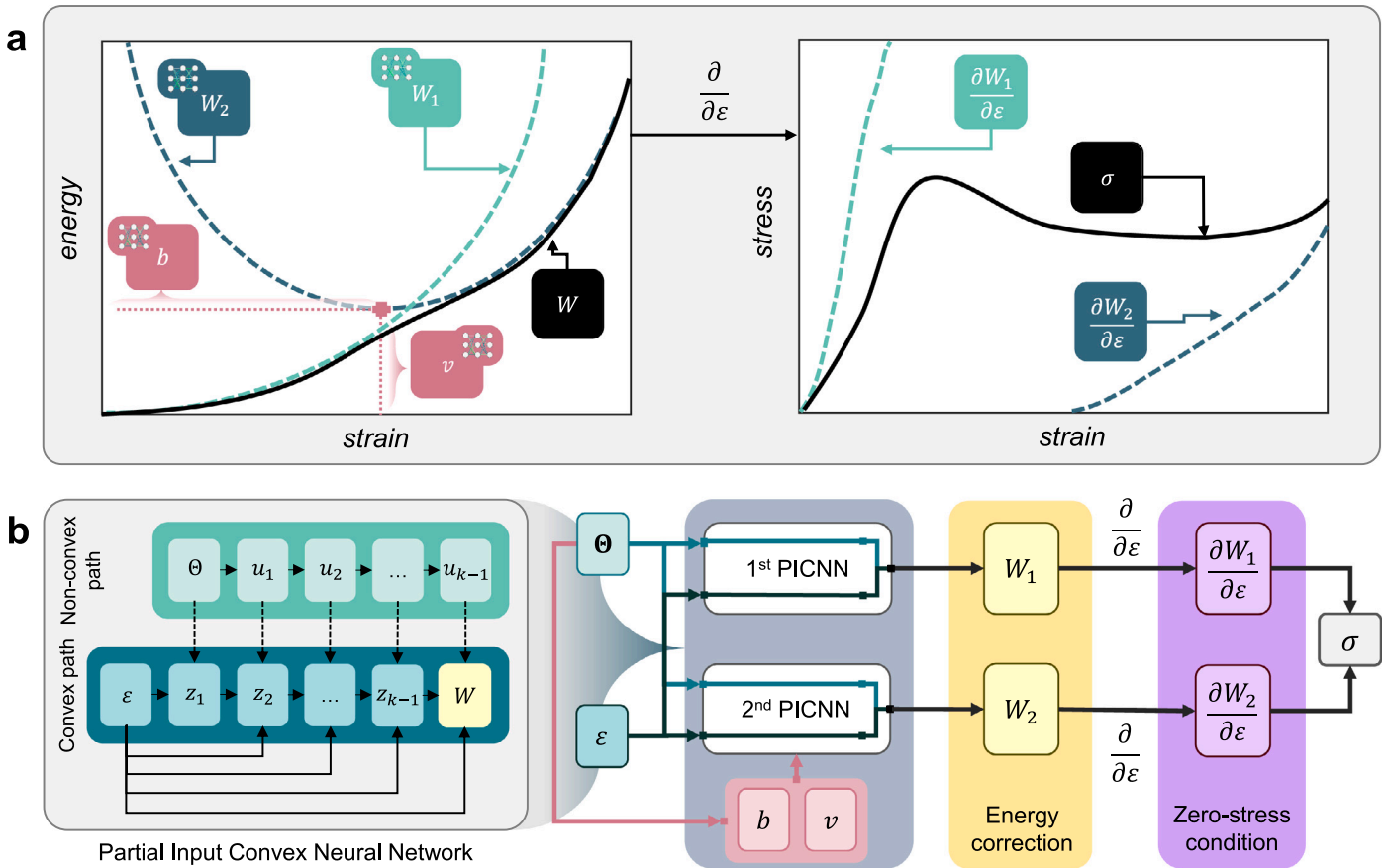
Fig. 3. Physics-enhanced deep learning framework. (**a**) *Left:* The uniaxial compression stress response (as a function of applied strain ε) of spinodal metamaterials is modeled as the derivative of a deep neural network-based nonconvex energy density potential $W(\varepsilon, \boldsymbol{\Theta})$. The model consists of two potentials convex in ε and given by separate neural networks: $W_1(\varepsilon, \boldsymbol{\Theta})$ with the energy and stress vanishing at $\varepsilon = 0$ by construction; $W_2(\varepsilon, \boldsymbol{\Theta})$ with the energy $(v + W_1(b, \boldsymbol{\Theta}))$ and vanishing stress at $\varepsilon = b$. Both $b(\boldsymbol{\Theta})$ and $v(\boldsymbol{\Theta})$ are also given by neural networks. The nonconvex potential W is obtained by a combination of W_1 and W_2. *Right:* The stress is obtained by differentiating W with respect to ε. Also shown are the derivatives of W_1, W_2 for reference. (**b**) Schematic of the partial input convex neural network (PICNN) architecture for W_1 and W_2 and their combination thereof. The PICNN architecture predicts an energy which is convex with respect to the strain ε (via convex path) and parameterized by the design parameters $\boldsymbol{\Theta}$ (via nonconvex path). See SI Appendix Machine Learning Framework section for details.

in bulk materials due to nonconvex energetics [45]. Therefore, we model a representative stress–strain response as the derivative of an underlying strain energy density potential, which in the context of uniaxial compression corresponds to the area under the stress–strain curve. Consequently, the potential as a function of the applied strain must be monotonically increasing while admitting nonconvexities to allow for instabilities. We emphasize that this notion of energy potential is used only as an inductive bias to facilitate the learning of the uniaxial stress–strain response; its physical and thermodynamical admissibility should not be considered as strictly as a homogenized constitutive model.

We model the above potential with a deep neural network (NN) $W(\varepsilon, \boldsymbol{\Theta})$ as a function of the applied strain ε and design parameters $\boldsymbol{\Theta}$. A classical NN based on, e.g., a multi-layer perceptron (MLP) architecture may not satisfy the constraint of monotonic increase with ε. Additionally, while such an NN is highly non-convex in its inputs by default, the degree of nonconvexity (loosely speaking) with respect to ε should be constrained. The experimental data clearly exhibits a single macrostructural instability, which should be accordingly reflected in the NN output. Since we are training on experimental data directly, the NN—left unchecked—can exhibit fine-scale but highly oscillatory behavior trying to fit the experimental noise [46]. To satisfy the above constraints, we introduce the following NN architecture.

We take inspiration from Kumar et al. [45] and combine multiple convex potentials into one monotonically increasing, but nonconvex potential as

$$W(\varepsilon, \boldsymbol{\Theta}) = \min\{W_1(\varepsilon, \boldsymbol{\Theta}), W_2(\varepsilon, \boldsymbol{\Theta})\}, \tag{3}$$

where $W_1(\varepsilon, \boldsymbol{\Theta})$ and $W_2(\varepsilon, \boldsymbol{\Theta})$ are two energy potentials that are convex in ε (but non-convex in $\boldsymbol{\Theta}$). However, the corresponding transition between phases is very sharp and not representative of the experimental data. Therefore, we relax Eq. (3) by allowing the phases to coexist with volume fractions $\gamma_1, \gamma_2 \in [0, 1]$, respectively, as

$$W(\varepsilon, \boldsymbol{\Theta}) = \min_{\gamma_1, \gamma_2} \left[\sum_{i=1}^{2} \gamma_i W_i(\varepsilon, \boldsymbol{\Theta}) - \underbrace{\left(-k_T \sum_{i=1}^{2} \gamma_i \log \gamma_i \right)}_{\text{configurational entropy}} \right], \tag{4}$$

with $\quad \gamma_1 + \gamma_2 = 1$.

Here, the configurational entropy (not physical entropy) penalizes the formation of phase mixtures. The constant $k_T > 0$ controls the influence of configurational entropy and in turn, the smoothness of the transition between W_1 and W_2. The effective stress σ is obtained as the derivative of $W(\varepsilon, \boldsymbol{\Theta})$

$$\sigma(\varepsilon, \boldsymbol{\Theta}) = \frac{\partial W}{\partial \varepsilon} = \frac{\exp\left(-\frac{W_1}{k_T}\right)}{\exp\left(-\frac{W_1}{k_T}\right) + \exp\left(-\frac{W_2}{k_T}\right)} \frac{\partial W_1}{\partial \varepsilon} + \frac{\exp\left(-\frac{W_2}{k_T}\right)}{\exp\left(-\frac{W_1}{k_T}\right) + \exp\left(-\frac{W_2}{k_T}\right)} \frac{\partial W_2}{\partial \varepsilon}, \tag{5}$$

where the simplification on the right side follows from the analytical solution of Eq. (4) (see Supplementary Note 3 for derivation).

We model the constituent potential W_1 as

$$W_1(\varepsilon, \boldsymbol{\Theta}) = \underbrace{\mathcal{P}_{\omega_1}(\varepsilon, \boldsymbol{\Theta})}_{\text{first convex potential}} - \underbrace{\mathcal{P}_{\omega_1}(0, \boldsymbol{\Theta})}_{\text{energy correction}} - \underbrace{\varepsilon \frac{\partial \mathcal{P}_{\omega_1}(0, \boldsymbol{\Theta})}{\partial \varepsilon}}_{\text{stress correction}}. \quad (6)$$

Here, $\mathcal{P}_{\omega_1}$ denotes a *partial input convex NN* (PICNN) (see Ref. [47]) for architectural details) with the trainable parameter set ω_1. Due to the inherent property of PICNNs, the predicted W_1 is modeled to be only convex with respect to ε, but can have any arbitrary functional relationships with $\boldsymbol{\Theta}$. We name these functional relationships within the PICNN architecture as the convex and nonconvex path (as seen in Fig. 3b). Following the principle that non-negative weighted sums of convex functions are convex, the convex path only contains linear transformations with non-negative weights and convex non-decreasing nonlinear activation functions. Supplementary Note 3 provides further details on the PICNN architecture used here. This PICNN-based approach allows us to obtain energies convex in ε but non-convexly parameterized by $\boldsymbol{\Theta}$. Note that, $\mathcal{P}_{\omega_1}$ is merely the convex output of the PICNN, and only when combined with the energy and stress corrections it becomes the potential W_1. These correction terms ensure that W_1 identically satisfies zero energy and zero stress (strain-derivative of W_1) at zero strain, i.e., $\varepsilon = 0$.

We model the constituent potential W_2 as

$$W_2(\varepsilon, \boldsymbol{\Theta}) = \underbrace{\mathcal{P}_{\omega_2}(\varepsilon - b, \boldsymbol{\Theta})}_{\text{second convex potential}} - \underbrace{\left(\mathcal{P}_{\omega_2}(0, \boldsymbol{\Theta}) - v - W_1(b, \boldsymbol{\Theta}) \right)}_{\text{energy correction}}$$

$$- \underbrace{(\varepsilon - b) \frac{\partial \mathcal{P}_{\omega_2}(0, \boldsymbol{\Theta})}{\partial \varepsilon}}_{\text{stress correction}}, \quad (7)$$

$$\text{with} \qquad b = \mathcal{F}_\beta(\boldsymbol{\Theta}) \geq 0 \qquad \text{and} \qquad v = \mathcal{F}_v(\boldsymbol{\Theta}) \geq 0.$$

Here, $\mathcal{P}_{\omega_2}$ denotes another PICNN that is convex in ε and contains trainable parameters ω_2. However, unlike W_1, the energy and stress correction terms ensure that minimizer and minimum of W_2 are non-zero, i.e., $\varepsilon = b \geq 0$ and $W_2(b, \boldsymbol{\Theta}) = v + W_1(b, \boldsymbol{\Theta})$. Both b and v act as offsets of W_2 with respect to W_1 in the strain vs. energy space. Their values are given by two additional classical MLP neural networks $\mathcal{F}_\beta(\boldsymbol{\Theta})$ and $\mathcal{F}_v(\boldsymbol{\Theta})$, parameterized by β and v, respectively. The non-negativity constrain on b and v ensure that the non-convex combination of W_1 and W_2 in Eq. (4) yields a W that is monotonically increasing (see Fig. 3a). Note that, while we limit the construction of W to a combination of only two convex potentials, additional potentials can be incorporated to model more complex nonlinear behavior if needed.

We represent the experimentally generated dataset as

$$\mathcal{D} = \left\{ \left(\boldsymbol{\Theta}^{(n)}, \tilde{\varepsilon}_t^{(n,i)}, \tilde{\sigma}_t^{(n,i)} \right) \; : \; t = 1, \ldots, T(n,i); \; i = 1, 2, 3; \; n = 1, \ldots, N \right\}, \quad (8)$$

where n, i, and t denote the different sample, direction of loading, and loadstep during loading, respectively. The total number of loadsteps $T(n,i)$ as well as change in strain between two loadsteps may not necessarily be the same across different samples and direction of loading. The forward ML model is then trained to minimize the mean absolute percentage error (MAPE) loss in stress predictions across the small training dataset:

$$\omega_1, \omega_2, \beta, v \; \leftarrow \; \underset{\omega_1, \omega_2, \beta, v}{\arg\min} \; \frac{1}{|\mathcal{D}|} \sum_{n=1}^{N} \sum_{i=1}^{3} \sum_{t=1}^{T(n,i)} \left| \frac{\sigma\left(\tilde{\varepsilon}_t^{(n,i)}, \boldsymbol{\Theta}^{(n)} \right) - \tilde{\sigma}_t^{(n,i)}}{\tilde{\sigma}_t^{(n,i)}} \right|. \quad (9)$$

The loss function computes the relative error of the predicted and target values, ensuring that the predictions are forced to be accurate, regardless of the magnitude of the target value. Supplementary Note 3 presents the detailed ML training protocols.

Due to the limited amount of data, we take additional measures to facilitate the training process. For each data point, we distinguish between the type of spinodal topologies (lamellar, columnar, or cubic)

and the loading directions ($\hat{e}_1$, $\hat{e}_2$, and $\hat{e}_3$). For each of these cases, we train distinct ML models and post-hoc lump them into a unified model; see Supplementary Note 3 for details.

We evaluate the predictive capabilities of our forward model by using test samples and their stress–strain data which the ML framework has not seen during training. In Fig. 4a, we present the predicted (teal) and the experimentally measured (red) stress–strain curves across three directions each for three test samples—one each of lamellar, columnar, and cubic topologies. We also plot the derivatives $\partial W_1/\partial\varepsilon$ (dark blue) and $\partial W_2/\partial\varepsilon$ (light blue) of the two convex constituent energy potentials for reference. We generally observe a MAPE of 6%–12% across all types of spinodal structures and across all directions. However, we do note that the errors for predicting the material behavior for lamellar structures (with small cone angles) are generally higher than for columnar and cubic structures. We attribute the higher errors for lamellar topologies to the relatively high sensitivity to imperfections and unpredictable localized mechanical behavior. Fig. 4b illustrates the accuracy of the models across all the nine test cases. Despite the small dataset, we observe a goodness-of-fit $R^2 > 0.96$ for stress, $R^2 > 0.99$ for absorbed energy (i.e., cumulative area under the curve), and $R^2 > 0.90$ for the incremental stiffness (i.e., slope of the curve) with respect to the experimental ground truth at every strain point. Supplementary Note 2 provides details on how the absorbed energy and incremental stiffness are computed. The lower accuracy in incremental stiffness predictions may be attributed to compounding of errors when computing derivatives of the stress–strain curve.

2.4. Morphology-dependent deformation mechanisms

To bridge the gap between the measured/predicted nonlinear responses and the morphology descriptors that lead to various deformation mechanisms, we employed nonlinear finite element models of selected morphologies within our design space. We leveraged these simulations to add mechanistic insight to performance metrics such as absorbed energy and its relation to our spinodal design space, particularly linking nonlinear mechanisms to surface-curvature distributions in a given morphology. As representative designs of the lamellar, columnar, and cubic categories, we selected the $\boldsymbol{\Theta} = [0°, 0°, 33°]$, $\boldsymbol{\Theta} = [0°, 23°, 37°]$, and $\boldsymbol{\Theta} = [20°, 23°, 24°]$ morphologies, respectively, to model within the finite element framework.

To match the conditions in our experiments, we modeled each morphology with no constrains on the lateral faces while the degrees of freedom of the bottom nodes were fully constrained and a compressive displacement was imposed on the top nodes. To capture complex responses while managing the computational cost of each simulation, we discretized each morphology using structural shell elements endowed with an elasto-plastic material model, where the plastic flow curve was determined from experiments on the constituent polymer (see Supplementary Note 2). We note that this plasticity model was chosen for its simple implementation, while it is not intended to fully capture the physical mechanisms of permanent deformation in our polymeric constituent material. As a result, we employ these simulations as a tool to uncover differences in deformation mechanisms between representative morphologies, instead of as an all-encompassing material model for our spinodal samples. Improvements to the model can be made to account for polymer-specific physics during nonlinear deformation [48–50], but are beyond the scope of this work.

Comparing the simulations to the *in situ* experimental responses demonstrated kinematic agreement within all morphologies, prior to the emergence of fracture events within the polymeric shells which were not intended to be captured numerically. For instance, simulations of compression on the $\boldsymbol{\Theta} = [0°, 23°, 37°]$ morphology along the $\hat{e}_1$ direction (Fig. 5a, other morphologies/directions in Figs. S13 and S14) captured nonlinear responses such as buckling and self-contact. In agreement with experiments, the simulations predicted the
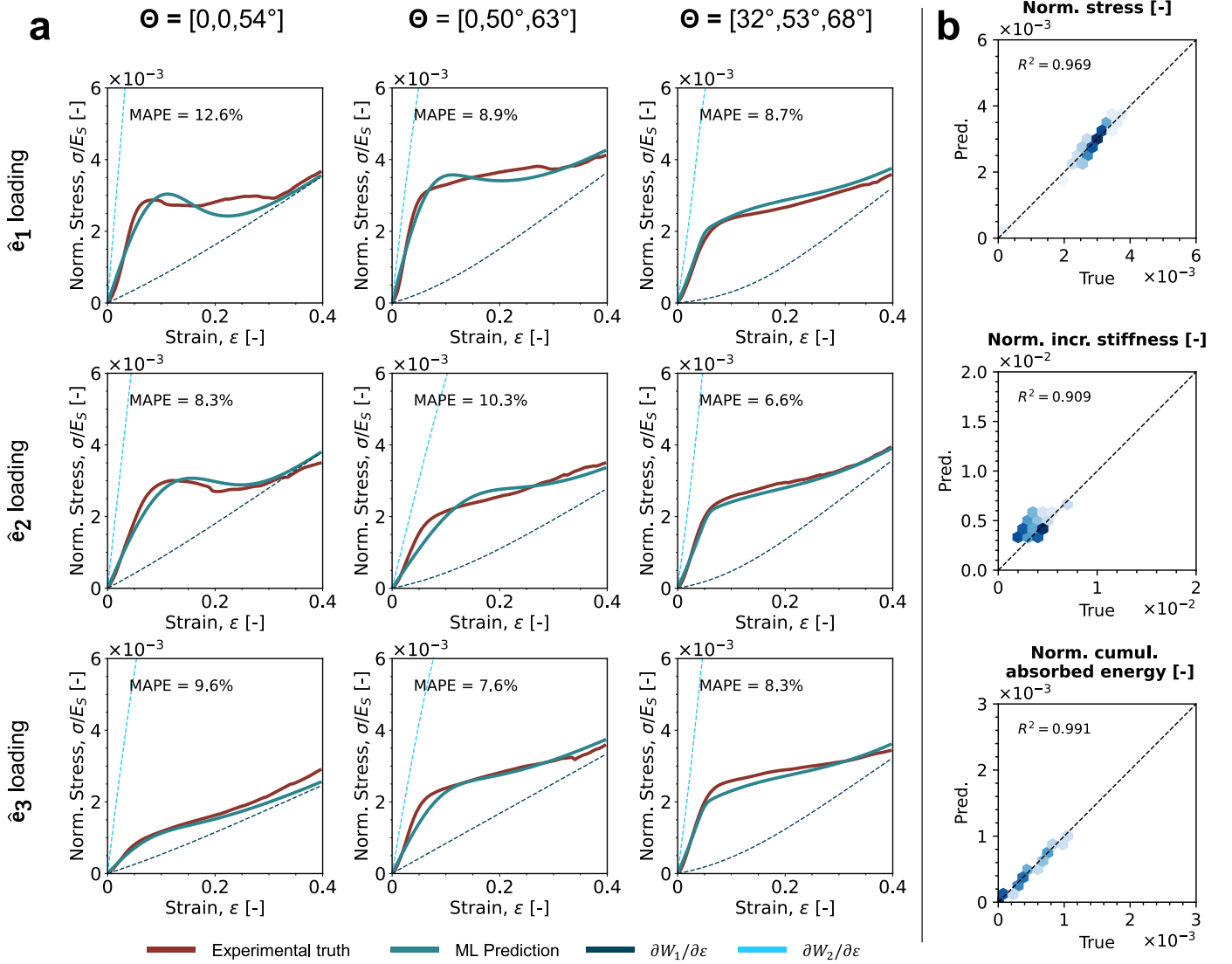
Fig. 4. Forward model results. (**a**) Stress–strain plots showing the ML-predicted (teal) versus experimental ground truth (red) curves for three representative spinodal design in the test dataset (i.e., sample from outside the training dataset). For reference, we show the derivatives $\partial W_1/\partial \varepsilon$ (dark blue) and $\partial W_2/\partial \varepsilon$ (light blue). (**b**) Distribution of predicted vs. ground truth values on the test dataset for normalized stress, normalized incremental stiffness (i.e., slope of curve), and normalized cumulative energy absorbed (i.e., area under curve) at all strain increments. The dashed line represents the ideal line with zero intercept and unit slop; R^2 denotes the corresponding goodness-of-fit.

buckling events to emerge at $\varepsilon \approx 5\%$, accompanied by stress localization at regions of higher curvature. Beyond the onset of buckling, the simulations predicted accumulation of equivalent plastic strain $\bar{\varepsilon}^p$ primarily in these high-curvature regions, agreeing with the sites of fracture initiation in *in situ* experiments. While the linear response of the simulations proved to be consistently stiffer than that observed in experiments (Fig. 5b), as expected in a model that is agnostic of minor fabrication defects, the qualitative large-deformation response was in agreement and the effective yield strength σ_y^* predictions were within 2% of experimental values—with the exception of the lamellar sample whose instability-driven onset of nonlinearity was within 13%. Most importantly, the simulations provided estimates of energy distributions within the probed morphologies, suggesting energy absorption to be primarily due to plastic dissipation. Specifically, plastic dissipation was calculated to equal the elastic strain energy in the structure at compressive strains of 5%, shortly after the emergence of buckling (Fig. 5c). Even at strains of 20% in various morphologies, contributions due to frictional dissipation were minimal, with plastic dissipation accounting for up to 78% of the total internal energy in the system in the case of the lamellar architecture.

To establish a connection between these local phenomena and the underlying geometry, we introduce a parameter η termed the normal participation factor (NPF), which denotes the degree to which surfaces within an architecture align with the loading direction. For a given surface element i, the NPF $0 \leq \eta_i \leq 1$ is computed as

$$\eta_i = 1 - (\mathbf{n_i} \cdot \mathbf{e_d})^2, \tag{10}$$

where $\mathbf{n_i}$ represents the normal vector of the element and $\mathbf{e_d}$ denotes the direction of loading. The NPF assumes a value of 1 when the loading direction is parallel to the surface, while it approaches 0 when the loading direction is perpendicular. Using these finite element models to determine plastic strain localization in spinodal morphologies reveals a correlation between the regions of high NPF and localized equivalent plastic strain for a majority of the morphologies (Fig. 5d (*i, ii*)).

However, some morphologies such as $\Theta = [0°, 0°, 33°]$ under loading in the $\hat{e}_1$ direction show a greatly diminished correlation between the distribution of $\bar{\varepsilon}_p$ and NPF values (Fig. 5d (*iii*)). This discrepancy arises due to buckling phenomena, which lead to less efficient load distribution upon buckling and corresponding localized strain accumulation.
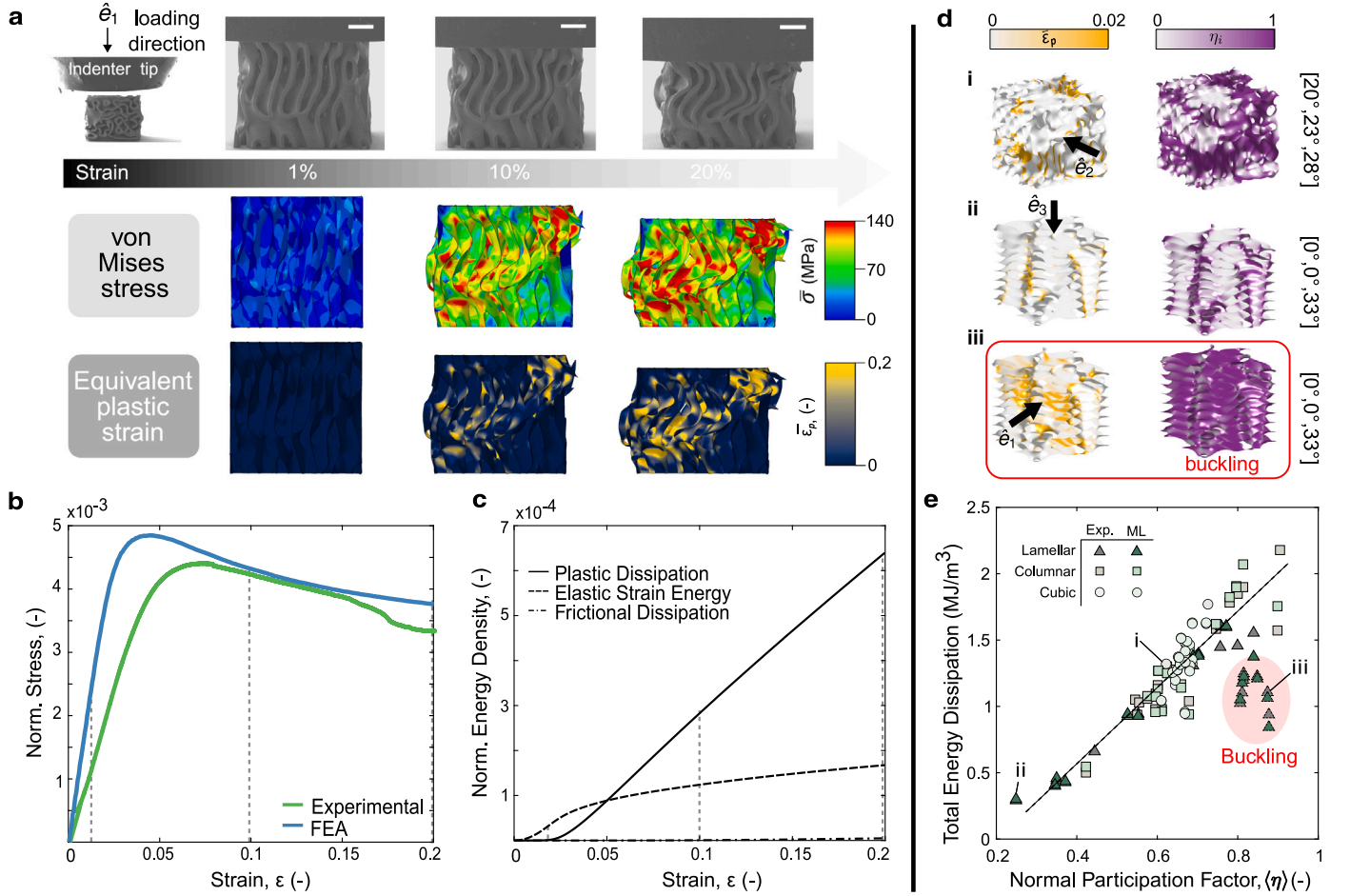
Fig. 5. Finite-strain simulations and normal participation factor. (**a**) *In situ* snapshots for $\Theta = [0°, 23°, 37°]$ loaded in the $\hat{e}_1$ direction for 1%, 10%, and 20% strain points. Scale bars, 10 μm. (**b**) Comparison between *in situ* experiment (green line) and simulation (blue line) for the normalized stress–strain curves up to 20% strain. (**c**) Distribution of energy mechanisms as a function of strain, including plastic dissipation, elastic strain energy, and frictional dissipation. Up to 20% strain, friction effects are negligible and plastic dissipation is the dominant mode of dissipation. (**d**) Geometric representations of the normal participation factor η (NPF) for three representative cases and its correlation to the equivalent plastic strain $\bar{\varepsilon}_p$. The dark purple regions correspond to $\eta \approx 1$ which correlates to regions that undergo high plastic deformation (gold regions) for the cases shown in (**i**) and (**ii**), while diminished correlation occurs in the buckling-prone case in (**iii**). (**e**) Linear correlation between total energy dissipation and η, where the data points corresponding to the three cases in (**d**) are indicated, showing a loss of correlation for geometries that undergo buckling events.

Using these local η_i values, we define the average NPF for the entire morphology under a given loading direction, denoted as $\langle \eta \rangle$, as

$$\langle \eta \rangle = \frac{\sum_{i=1}^{N} A_i \eta_i}{\sum_{i=1}^{N} A_i}, \tag{11}$$

where A_i represents the area of the ith surface element. Plotting the $\langle \eta \rangle$ against the total energy dissipation, as shown in Fig. 5e, reveals a notable correlation between the two parameters. Points deviating from the trend line, signify exceptional cases characterized by buckling instabilities. Furthermore, the predictions obtained from the ML algorithm are overlaid, demonstrating its ability to capture the large-deformation effect of these instabilities. These exceptional cases underscore the necessity for employing two potentials within the ML framework, rather than one, to accurately represent the nonlinear energy absorption associated with buckling phenomena in spinodal metamaterials.

2.5. Inverse design for tailored stress–strain response

We use the predictive capabilities of the forward ML model as a fast surrogate (to experiments) for inverse designing spinodal metamaterials with prescribed nonlinear stress–strain responses. Let $S = \{(\hat{\varepsilon}_t, \hat{\sigma}_t) : t = 1, \ldots, T\}$ be a desired stress–strain response for quasi-static uniaxial

compression loading discretized on $T > 0$ points. We use the MAPE loss to formulate the inverse design task as an optimization:

$$\Theta, \hat{e}_i \leftarrow \underset{\Theta, \ i \in \{1,2,3\}}{\arg\min} \ \frac{1}{T} \sum_{t=1}^{T} \left| \frac{\sigma^{(i)}(\hat{\varepsilon}_t, \Theta) - \hat{\sigma}_t}{\hat{\sigma}_t} \right|. \tag{12}$$

The stress predictions $\sigma^{(i)}(\hat{\varepsilon}_t, \Theta)$ are evaluated via the forward ML model using Eq. (5), where the superscript $(\cdot)^{(i)}$ indicates the model corresponding to the $\hat{e}_i$ principal direction of loading (see Supplementary Note 4). This has two advantages: *(i)* the numerical optimization requires several evaluations of the stress predictions, which are orders of magnitude faster when performed via the forward ML model than experiments (or even simulations, if feasible); and *(ii)* while gradient-based optimization schemes (e.g., gradient descent [51]) are more efficient and stable than non-gradient-based methods, they require computing the sensitivities of the loss function with respect to the variables (in this case, Θ). Leveraging the computational graph and backpropagation of NNs [52], the forward ML model enables automatic differentiation [53] of the stress predictions σ and in turn, the loss function with respect to Θ. In contrast to computationally intensive numerical differentiation (perturbing the input and re-evaluating the loss function), automatic differentiation provides analytically exact gradients and enables a more stable numerical optimization.

Here, we use the Adaptive Moment Estimation or Adam [54] optimizer to solve Eq. (12). We highlight that the inverse design challenge
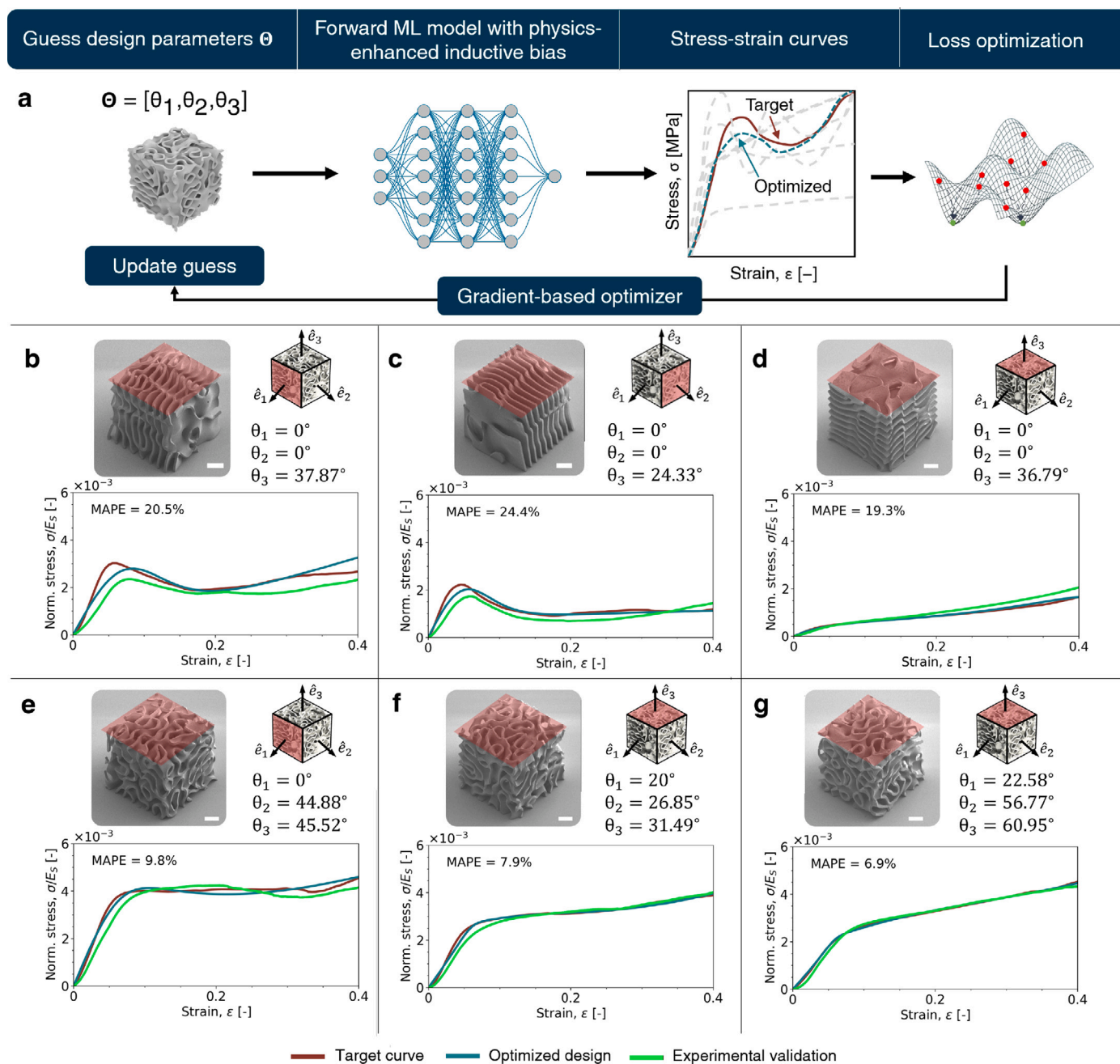
Fig. 6. Inverse design framework and results. (**a**) The forward ML model serves as a fast surrogate to evaluate the structure–property relations within an iterative gradient-based optimization scheme. The input to the optimization scheme is a target stress–strain curve (indicated in dark red) and the optimizer iteratively updates the guess design parameters until the loss between the predicted and target stress–strain curve is minimized. (**b–g**) We optimize for six target stress–strain curves which are not present in the training dataset. The stress–strain plots show the target (red), optimized (teal), and experimentally verified (green) stress–strain curves (with MAPEs specified between the target and experimental response). For each optimized design, we provide SEM images of the fabricated samples corresponding to the optimized design parameters and the green stress–strain curves, with the vertical direction (substrate normal direction) corresponding to the loading direction highlighted in transparent red. A graphic of the standard axis convention used in this work is given for reference to loading direction. Scale bar, 20 µm.

is ill-posed as multiple designs can exhibit the target stress–strain curve. To bypass this challenge, we perform the optimization for different initial guesses in parallel and select the design with the lowest MAPE loss at the end. Supplementary Note 4 provides additional details on the optimization protocols and a pseudocode is presented in Supporting Information Algorithm 1.

To demonstrate the efficacy of our framework, we inversely design for target curves from our dataset, which were multiplied by a factor $\kappa = 1.2$. This ensures that the target mechanical behavior is beyond the stress–strain curves which were provided in the limited experimentally generated training set. We present six distinct target curves and their

respective optimization results in Fig. 6b–g. For each target (red) we optimize the design parameters to obtain an optimized curve (teal). Finally, we fabricate and test the structures with the optimized design parameters (green). The first target (b) has a strain-softening region with a subsequent strain-hardening region. Similarly, the second target (c) has a strain-softening region after the yield point, with a subsequent stress plateau. The third target (d) exhibits a relatively stable, monotonically increasing stress–strain response. The fourth target (e) has a relatively stiff linear elastic region and a subsequent stress plateau. The last two targets (f–g) possess stiff linear elastic region with a subsequent strain-hardening behavior. For the first target, the optimization

framework proposes a lamellar structure ($\boldsymbol{\Theta} = [0°, 0°, 37.87°]$) oriented along the $\hat{e}_1$-direction to achieve the desired stress–strain response. The fabricated structure matches the target response well with a MAPE of 20.5%. Most of the error stems from the strain-hardening region in the fabricated sample, which the forward module overpredicted. For the second target, also a lamellar structure ($\boldsymbol{\Theta} = [0°, 0°, 24.33°]$) oriented along the $\hat{e}_2$-direction, the MAPE between the response of the fabricated sample and the target curve is 24.4%. Here, the three curves are offset by a small margin, with the experiment showing a lower yield stress than what was predicted. The overall macro-structural behavior, i.e., strain-softening with subsequent stress-plateau, is captured with the optimized design parameters. For the third target, the fabricated target ($\boldsymbol{\Theta} = [0°, 0°, 36.97°]$) exhibits a higher strain-hardening than what was predicted, which results in an MAPE of 19.3%. However, similarly to the second target, the overall stress–strain behavior is well captured. For the fourth, fifth, and sixth target, we observe excellent matches with the queried stress–strain behaviors with MAPEs of $9.8\%, 7.9\%$, and 6.9%.

Validation on these six target curves demonstrates the validity of the proposed framework to capture highly nonlinear responses in complex 3D spinodal morphologies that lack internal symmetry and periodicity. While quantitative agreement between experimental and optimized responses improves for columnar or cubic morphologies, which have less pronounced anisotropy compared to lamellar ones, the overall qualitative response was always captured regardless of the $\boldsymbol{\Theta}$ representation. Specifically, in scenarios where micromechanical instabilities or failure events led to a macro-structural negative-stiffness responses, the framework accurately accounted for these features with closely matched stress and strain levels. Moreover, our results demonstrate that the physical basis chosen for our ML framework accurately describes the structural response of thick-shell spinodal structures at finite strains—a complex parameter space that is intractable to fully explore through high-fidelity simulations or experiments alone.

3. Conclusion

Designing complex spinodal metamaterials with a wide range of topologies and corresponding nonlinear mechanical behavior is challenging, especially when computational modeling can be costly and experimental data is scarce. Here, we introduced a physics-enhanced ML and optimization framework that bypasses this challenge by directly using extremely sparse experimental data and enables the inverse design of spinodal structures with tailored finite-strain mechanical responses.

The lack of data for learning the structure–property relations is compensated by the physics-based inductive biases, which aid in identifying nonlinear responses such as instability- and localization-dominated responses. Tracing these responses back to nonconvex energetic potentials allows for a versatile framework that may be applied to a variety of lightweight microstructures employed in the mechanical metamaterials community. Inspired from phase transformation modeling approaches, combining multiple convex (in strain) neural networks to form nonconvex but monotonically increasing potentials can accurately and efficiently capture complex nonlinear stress–strain behavior in presence of extreme and localized deformation including failure. At the same time, partial input convexity of PICNN architectures allows capturing arbitrary non-convex functional relations with the design parameters of spinodal metamaterials.

In this work, we verify the importance of the above described physical basis in the ML model by looking at the local deformation mechanisms. Despite the computational cost, finite element simulations lead to the observation that plastic dissipation is the dominant dissipation mechanism up to 20% strain and thus the geometric normal participation factor η was introduced to describe the local loading distributions. We found that η and energy absorption of our samples were strongly correlated. However, the samples that underwent buckling had diminished correlation due to premature strain localization, evidencing the physics-enhanced ML as a route to accurately and efficiently predict energy absorption for these unique cases and highlighting the need for the second convex energy well.

While we developed this framework for spinodal metamaterials, we note that the ML inverse design framework and the physics-enhanced inductive biases are sufficiently general to be individually (or in combination) adapted for any class of metamaterials (e.g., truss, plate, or TPMS lattices) and data representation (e.g., vector, graph, or pixel/voxel-based parameterizations). The gradient-based optimization strategy can also be adapted to other approaches, including generative ML methods such as conditional variational autoencoders [42] and diffusion models [33]. Additionally, one could extend the model to handle multiple loading cycles by introducing internal parameters that track the material's loading history. These additions could allow the model to distinguish between loading and unloading phases and account for plastic deformation. Moreover, in this work, we have performed a comparatively large number of high-fidelity experiments in both *ex situ* and *in situ* formats to assist in the understanding of the ML model from a physical basis. We leverage microscale fabrication and testing to obtain an order-of-magnitude increase in data throughput—necessary for applicability to ML frameworks. From the 321 experiments performed in this work, we were able to capture the complex localized deformation behavior of thick-shelled spinodal morphologies, which led us to gain critical insights into the physical interpretation of our ML framework results. This work closes the gap in understanding the structure-to-property relations of high-relative-density spinodal morphologies at finite strains, relevant to applications of high energy absorption materials. The combination of using an experimentally generated dataset to train a physics-enhanced ML framework shows a promising avenue for designing and understanding the complex architected materials of the future.

Methods

Details on the virtual specimen generation (Supplementary Note 1), the sample fabrication, physical characterization and finite element analysis (Supplementary Note 2), the data preprocessing, the PICNN-framework and training protocols (Supplementary Note 3), and the gradient-based multi-initialization optimization scheme (Supplementary Note 4) are provided in the Supplementary information.

CRediT authorship contribution statement

Prakash Thakolkaran: Writing – review & editing, Writing – original draft, Visualization, Software, Methodology, Investigation, Formal analysis, Data curation. **Michael Espinal:** Writing – review & editing, Writing – original draft, Visualization, Validation, Investigation, Formal analysis, Data curation. **Somayajulu Dhulipala:** Writing – review & editing, Writing – original draft, Formal analysis, Data curation. **Siddhant Kumar:** Writing – review & editing, Supervision, Resources, Funding acquisition, Conceptualization. **Carlos M. Portela:** Writing – review & editing, Supervision, Resources, Funding acquisition, Conceptualization.

Declaration of competing interest

The authors declare the following financial interests/personal relationships which may be considered as potential competing interests: Carlos Portela reports financial support was provided by National Science Foundation. If there are other authors, they declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

Acknowledgments

Appendix A. Supplementary data

Supplementary material related to this article can be found online at https://doi.org/10.1016/j.eml.2024.102274.

Data and code availability

The data and codes generated in the current study are freely available at https://github.com/mmc-group/finite-strain-inverse-designed-spinodoids.

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